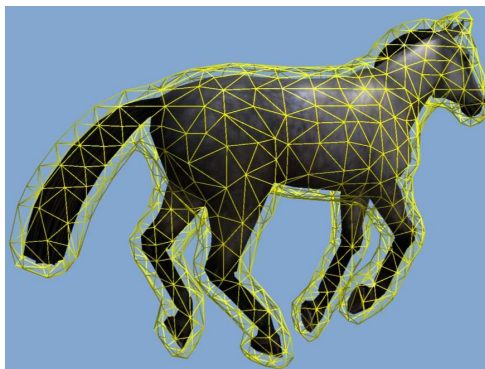


Bremen



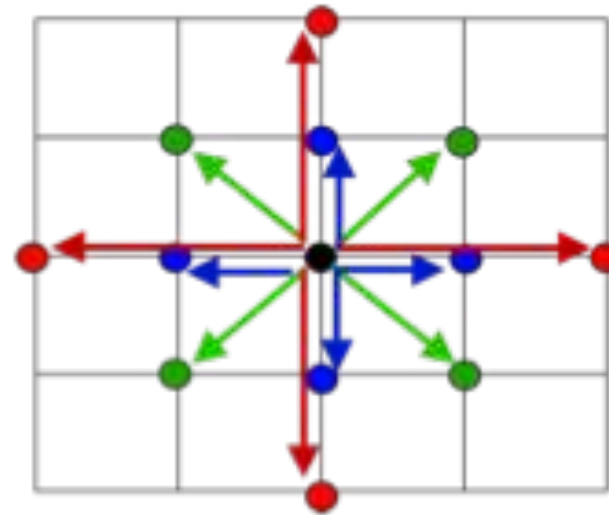
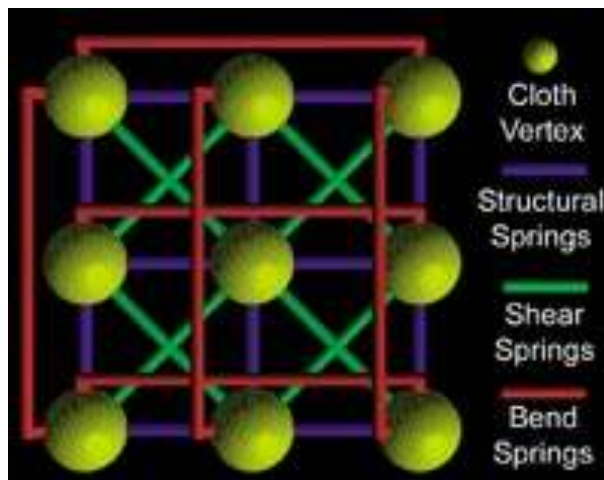
# Virtual Reality & Physically-Based Simulation Mass-Spring-Systems



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# Definition

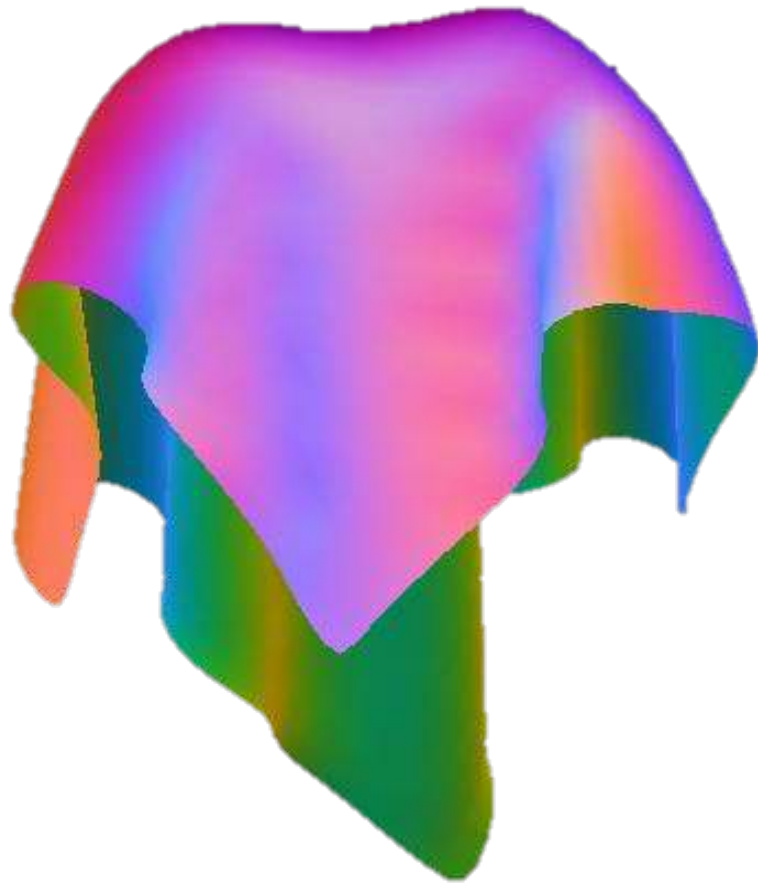
- A **mass-spring system** is a particle system consisting of:
  1. A set of point masses  $m_i$  with positions  $\mathbf{x}_i$  and velocities  $\mathbf{v}_i$ ,  $i = 1 \dots n$  ;
  2. A set of springs  $s_{ij} = (i, j, k_s, k_d)$  , where  $s_{ij}$  connects masses  $i$  and  $j$ , with rest length  $l_0$  , spring constant  $k_s$  (= stiffness) and the damping coefficient  $k_d$
- Typical spring topology:



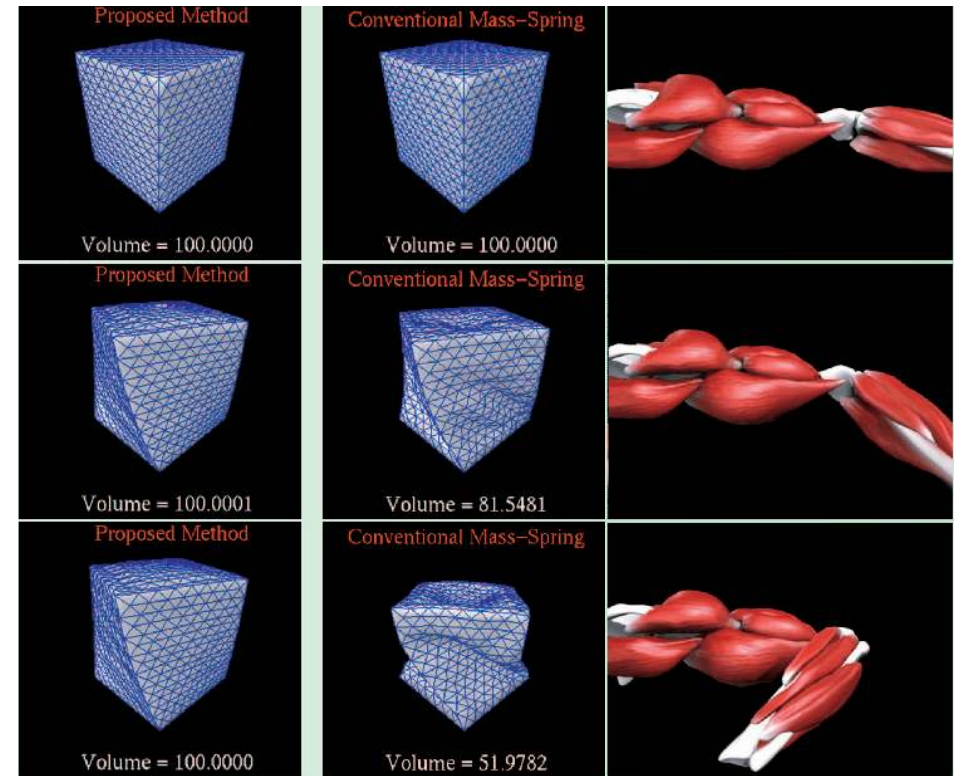
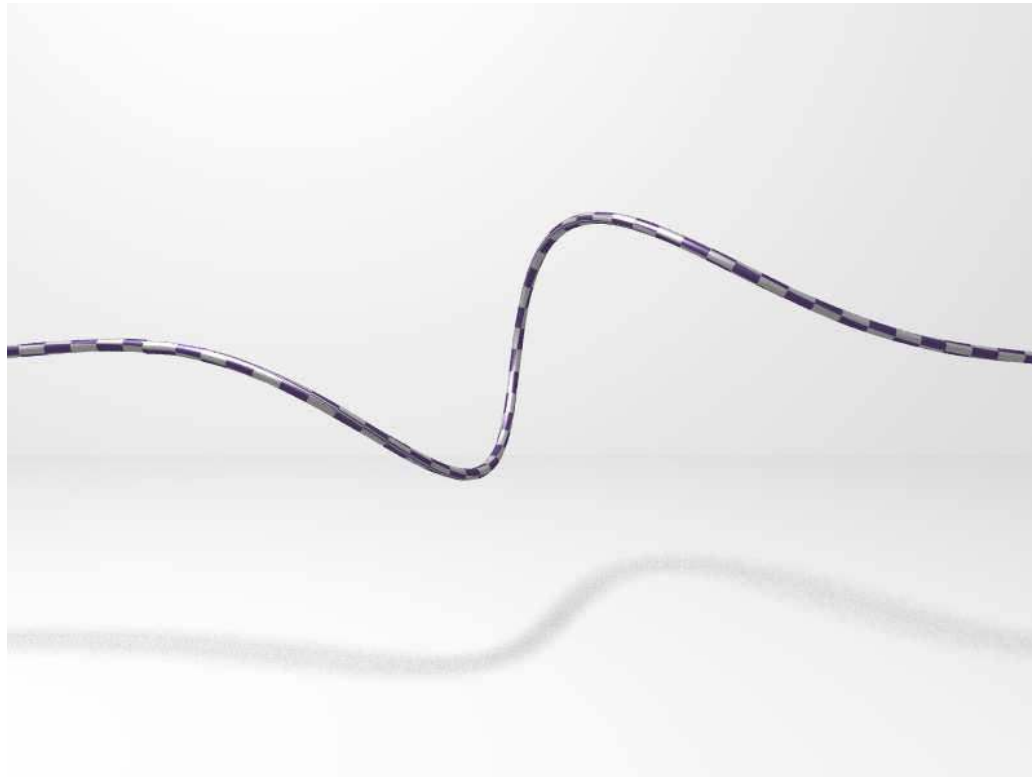
# Some Properties

- Advantages:
  - Very easy to program
  - Ideally suited to study different kinds of solving methods
  - Ubiquitous in games (cloths, capes, sometimes also for deformable objects)
- Disadvantages:
  - Some parameters (in particular the spring constants) are not obvious, i.e., difficult to derive
  - No built-in volumetric effects (e.g., preservation of volume)

# Example Mass-Spring System: Cloth



# Occasionally also Used for 1D and 3D Objects



# Did You Learn About Springs in Your Physics Class in School ?



Also, how many of you are familiar with vector calculus?

<https://www.menti.com/1io1dqhgvtv>

# Forces Exerted by a Single Spring (Plus Damper)

- Given: masses  $m_i$  and  $m_j$  with positions  $\mathbf{x}_i$ ,  $\mathbf{x}_j$
- Let  $\mathbf{r}_{ij} = \frac{\mathbf{x}_j - \mathbf{x}_i}{\|\mathbf{x}_j - \mathbf{x}_i\|}$
- The force between particles  $i$  and  $j$ :

1. Force exerted by the spring (Hooke's law):

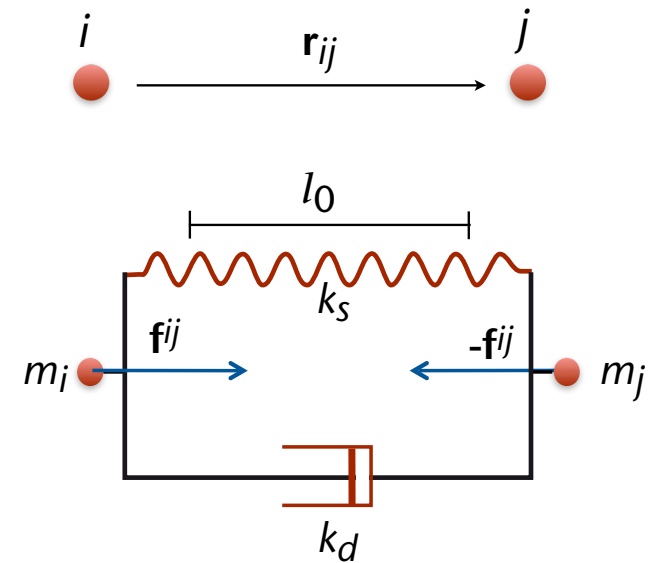
$$\mathbf{f}_s^{ij} = k_s \mathbf{r}_{ij} (\|\mathbf{x}_j - \mathbf{x}_i\| - l_0)$$

acts on particle  $i$  in the direction of  $j$

2. Force exerted on  $i$  by damper:  $\mathbf{f}_d^{ij} = -k_d ((\mathbf{v}_i - \mathbf{v}_j) \cdot \mathbf{r}_{ij}) \mathbf{r}_{ij}$

3. Total force on  $i$ :  $\mathbf{f}^{ij} = \mathbf{f}_s^{ij} + \mathbf{f}_d^{ij}$

4. Force on  $m_j$ :  $\mathbf{f}^{ji} = -\mathbf{f}^{ij}$



# Remarks

- A spring-damper element in reality:



- Alternative spring force: 
$$\mathbf{f}_s^{ij} = k_s \mathbf{r}_{ij} \frac{\|\mathbf{x}_j - \mathbf{x}_i\| - l_0}{l_0}$$
- Notice: from (4) it follows that the **total momentum is conserved**
  - Momentum  $\mathbf{p} = \mathbf{v} \cdot m$
  - Fundamental physical law (follows from Newton's laws)
- Note on terminology:
  - English "momentum" = German "Impuls" = velocity × mass
  - English "Impulse" = German "Kraftstoß" = force × time



# Simulation of a Single Spring

- From Newton's law, we have:  $\ddot{\mathbf{x}} = \frac{1}{m}\mathbf{f}$
- Convert this differential equation (ODE) of order 2 into ODE of order 1:

$$\dot{\mathbf{x}}(t) = \mathbf{v}(t)$$

$$\dot{\mathbf{v}}(t) = \frac{1}{m}\mathbf{f}(t)$$

- Initial values (boundary values):  $\mathbf{v}(t_0) = \mathbf{v}_0$  ,  $\mathbf{x}(t_0) = \mathbf{x}_0$
- By Taylor expansion we get:  $\mathbf{x}(t + \Delta t) = \mathbf{x}(t) + \Delta t \dot{\mathbf{x}}(t) + O(\Delta t^2)$
- Analogously:  $\mathbf{v}(t + \Delta t) = \mathbf{v}(t) + \Delta t \dot{\mathbf{v}}(t)$
- This integration scheme is called **explicit Euler integration**
- "Simulation" = "Integration of ODE's over time"

# The Main Loop for a Mass-Spring System

```

forall particles  $i$  :
    initialize  $\mathbf{x}_i, \mathbf{v}_i, m_i$ 
loop forever:
    forall particles  $i$  :

        
$$\mathbf{f}_i \leftarrow \mathbf{f}^g + \mathbf{f}_i^{coll} + \sum_{j, (i,j) \in S} \mathbf{f}(\mathbf{x}_i, \mathbf{v}_i, \mathbf{x}_j, \mathbf{v}_j)$$


        forall particles  $i$  :

            
$$\mathbf{v}_i += \Delta t \cdot \frac{\mathbf{f}_i}{m_i}$$

            
$$\mathbf{x}_i += \Delta t \cdot \mathbf{v}_i$$


    render the system every  $n$ -th time
    
```

$\mathbf{f}^g$  = gravitational force

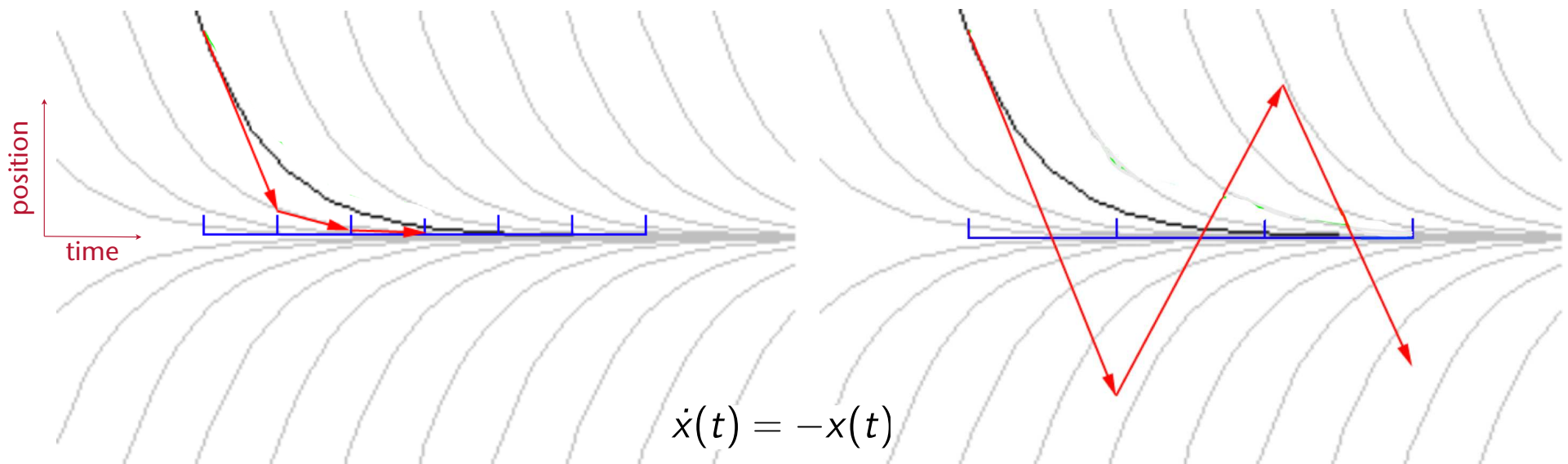
$\mathbf{f}^{coll}$  = penalty force exerted by collision (e.g., from obstacles)

# Properties of Explicit Euler Integration

- Advantages:
  - Can be implemented very easily
  - Fast execution per time step
  - Is "trivial" to parallelize on the GPU (→ "Massively Parallel Algorithms")
- Disadvantages:
  - Stable only for very small time steps
    - Typically  $\Delta t \approx 10^{-4} \dots 10^{-3}$  sec!
  - With large time steps, additional energy is generated "out of thin air", until the system explodes 😊
  - Example: overshooting when simulating a single spring
  - Errors accumulate quickly

# Example for the Instability of Euler Integration

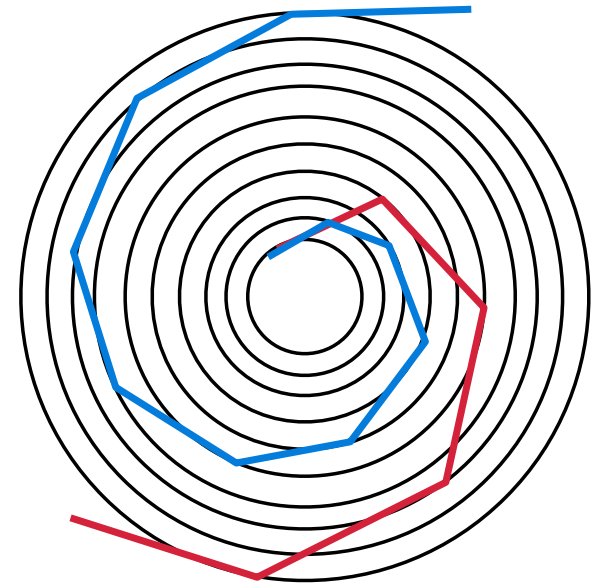
- Consider the differential equation  $\dot{x}(t) = -kx(t)$
- The exact solution:  $x(t) = x_0 e^{-kt}$
- Euler integration does this:  $x^{t+1} = x^t + \Delta t(-kx^t)$
- Case  $\Delta t > \frac{1}{k}$  :  $x^{t+1} = x^t \underbrace{(1 - k\Delta t)}_{<0}$   
 $\Rightarrow x^t$  oscillates about 0, but approaches 0 (hopefully)
- Case  $\Delta t > \frac{2}{k}$  :  $\Rightarrow x^t \rightarrow \infty !$



- Terminology: if  $k$  is large  $\rightarrow$  the ODE is called "*stiff*"
  - The stiffer the ODE, the smaller  $\Delta t$  has to be!

# Visualization of Error Accumulation

- Consider this ODE:  $\dot{\mathbf{x}}(t) = \begin{pmatrix} -x_2 \\ x_1 \end{pmatrix}$
- Exact solution:  $\mathbf{x}(t) = \begin{pmatrix} r \cos(t + \phi) \\ r \sin(t + \phi) \end{pmatrix}$
- The solution by Euler integration moves in spirals outward, no matter how small  $\Delta t$ !
- Conclusion: Euler integration accumulates errors, no matter how small  $\Delta t$ !

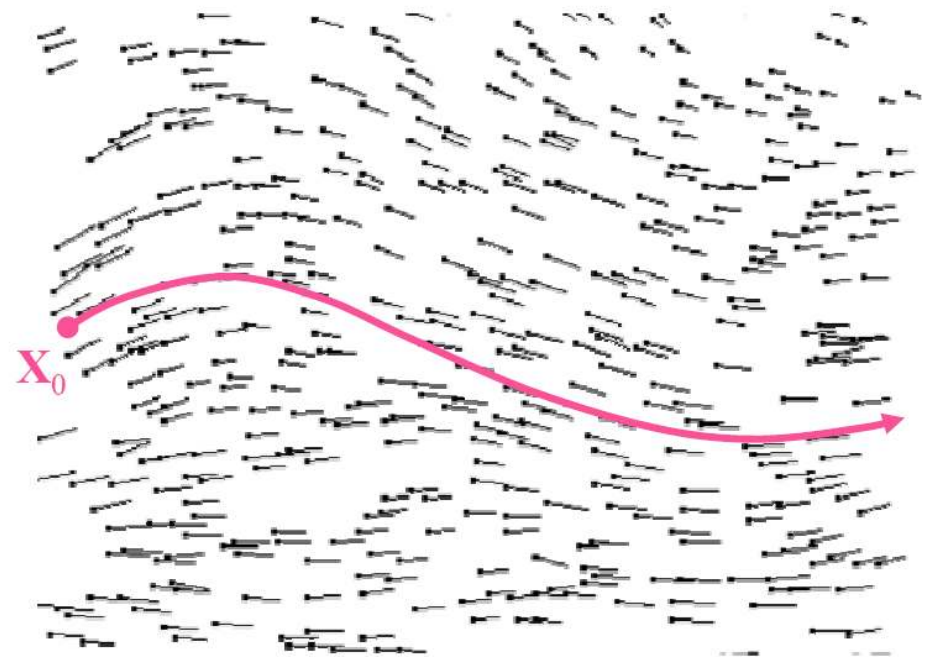


# Visualization of Differential Equations

- The general form of an ODE (ordinary differential equation):

$$\dot{\mathbf{x}}(t) = \mathbf{f}(\mathbf{x}(t), t)$$

- Visualization of  $\mathbf{f}$  as a vector field:
  - Notice: this vector field can vary over time!
- Solution of a boundary value problem = path through this field



# Other Explicit Integrators

- Runge-Kutta of order 2:
  - Idea: approximate  $\mathbf{f}(\mathbf{x}(t), t)$  by using the derivative at positions  $\mathbf{x}(t)$  and  $\mathbf{x}(t + \frac{1}{2}\Delta t)$
  - The integrator (w/o proof):

$$\mathbf{a}_1 = \mathbf{v}^t$$

$$\mathbf{a}_2 = \frac{1}{m} \mathbf{f}(\mathbf{x}^t, \mathbf{v}^t)$$

$$\mathbf{b}_1 = \mathbf{v}^t + \frac{1}{2} \Delta t \mathbf{a}_2$$

$$\mathbf{b}_2 = \frac{1}{m} \mathbf{f}\left(\mathbf{x}^t + \frac{1}{2} \Delta t \mathbf{a}_1, \mathbf{v}^t + \frac{1}{2} \Delta t \mathbf{a}_2\right)$$

$$\mathbf{x}^{t+1} = \mathbf{x}^t + \Delta t \mathbf{b}_1$$

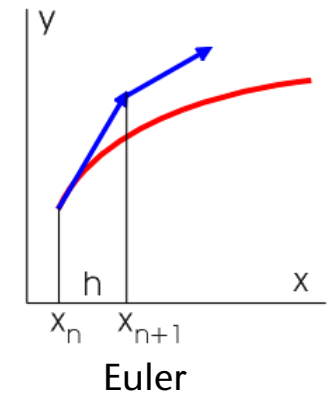
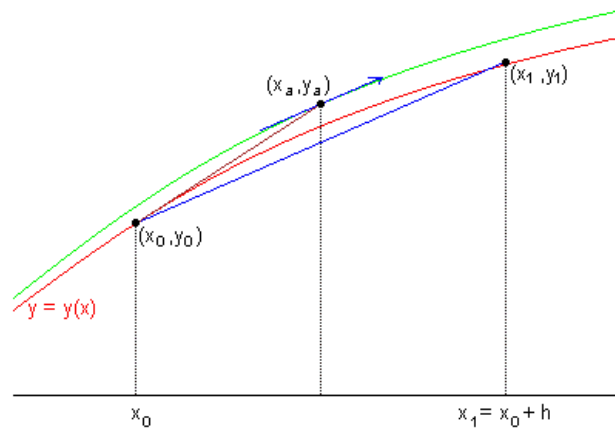
$$\mathbf{v}^{t+1} = \mathbf{v}^t + \Delta t \mathbf{b}_2$$

- Runge-Kutta of order 4:
  - **The** standard integrator among the explicit integration schemata
  - Needs 4 function evaluations (i.e., force computations) per time step
  - Order of convergence is:  $e(\Delta t) = O(\Delta t^4)$

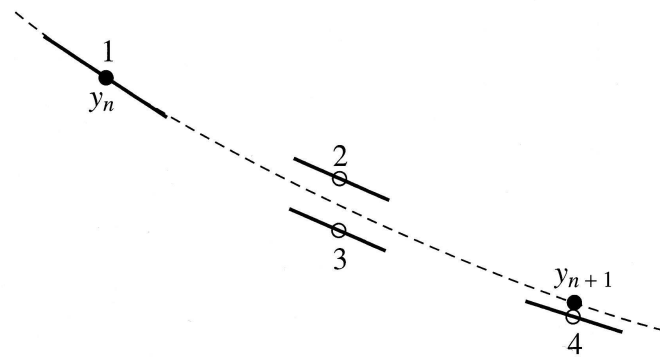


# Visualization

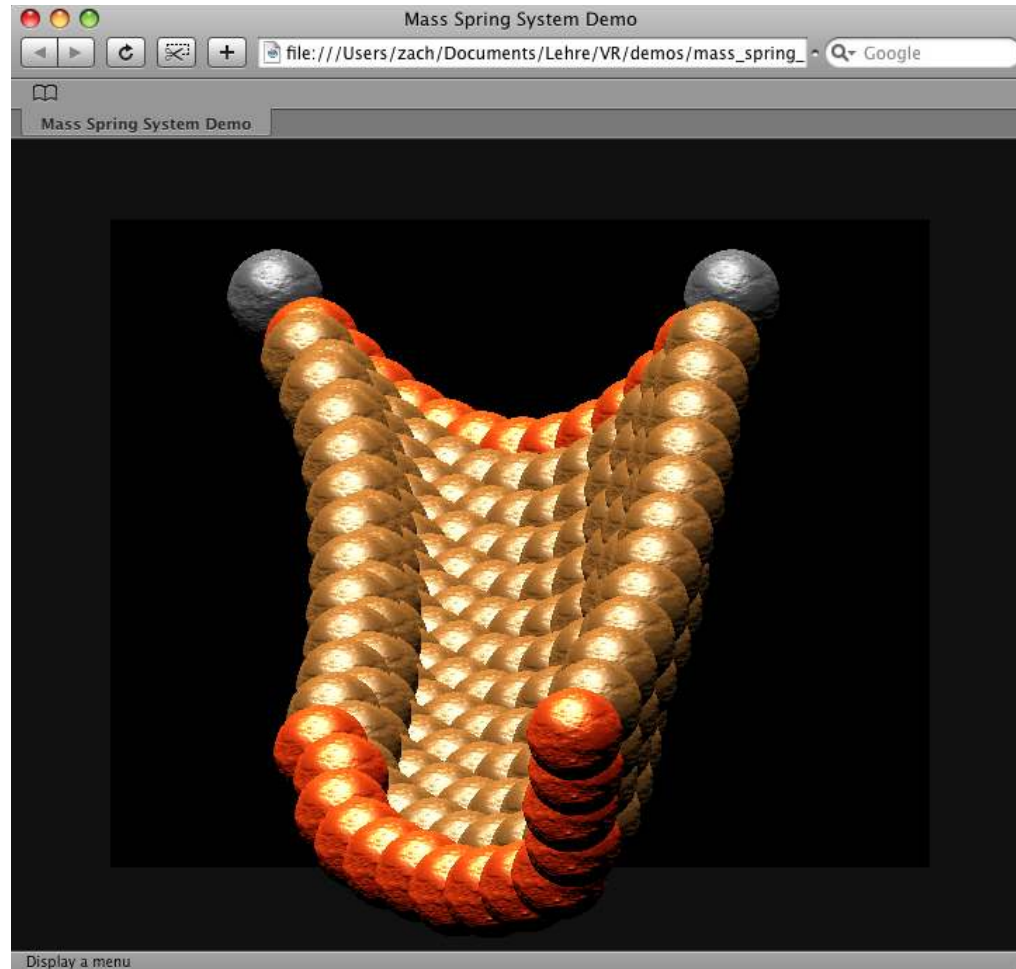
- Runge-Kutta of order 2:



- Runge-Kutta of order 4:



# Demo



<http://www.dhteumeuleu.com/dhtml/v-grid.html>

# How Does the Energy of a Mass-Spring System Change Over Time?



<https://www.menti.com/1io1dqhgvtv>

# Verlet Integration

- A general, alternative idea to increase the order of convergence: utilize values from the **past**
- Verlet integration = utilize  $\mathbf{x}(t - \Delta t)$
- Derivation:
  - Develop the Taylor series in both time directions:

$$\mathbf{x}(t + \Delta t) = \mathbf{x}(t) + \Delta t \dot{\mathbf{x}}(t) + \frac{1}{2} \Delta t^2 \ddot{\mathbf{x}}(t) + \frac{1}{6} \Delta t^3 \dddot{\mathbf{x}}(t) + O(\Delta t^4)$$

$$\mathbf{x}(t - \Delta t) = \mathbf{x}(t) - \Delta t \dot{\mathbf{x}}(t) + \frac{1}{2} \Delta t^2 \ddot{\mathbf{x}}(t) - \frac{1}{6} \Delta t^3 \dddot{\mathbf{x}}(t) + O(\Delta t^4)$$

- Add both:

$$\mathbf{x}(t + \Delta t) + \mathbf{x}(t - \Delta t) = 2\mathbf{x}(t) + \Delta t^2 \ddot{\mathbf{x}}(t) + O(\Delta t^4)$$

$$\mathbf{x}(t + \Delta t) = 2\mathbf{x}(t) - \mathbf{x}(t - \Delta t) + \Delta t^2 \ddot{\mathbf{x}}(t) + O(\Delta t^4)$$

- Initialization:

$$\mathbf{x}(\Delta t) = \mathbf{x}(0) + \Delta t \mathbf{v}(0) + \frac{1}{2} \Delta t^2 \left( \frac{1}{m} \mathbf{f}(\mathbf{x}(0), \mathbf{v}(0)) \right)$$

- Remark: the velocity does not occur any more! (at least, not explicitly)

# Constraints

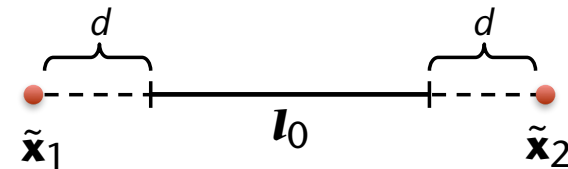
- Big advantage of Verlet over Euler & Runge-Kutta: makes it very easy to handle constraints on positions
- Definition: **constraint** = a condition on the position of one or more mass points
- Examples:
  1. A point must not penetrate an obstacle
  2. The distance between two points must be constant, or distance must be  $\leq$  some maximal distance

- Example: consider the constraint  $\|\mathbf{x}_1 - \mathbf{x}_2\| \stackrel{!}{=} l_0$
- 1. Perform one Verlet integration step  $\rightarrow \tilde{\mathbf{x}}^{t+1}$  (tentative new positions)
- 2. Enforce the constraint:

$$d = \frac{1}{2}(\|\tilde{\mathbf{x}}_2^{t+1} - \tilde{\mathbf{x}}_1^{t+1}\| - l_0)$$

$$\mathbf{x}_1^{t+1} = \tilde{\mathbf{x}}_1^{t+1} + d\mathbf{r}_{12}$$

$$\mathbf{x}_2^{t+1} = \tilde{\mathbf{x}}_2^{t+1} - d\mathbf{r}_{12}$$



- Problem: if several constraints are to constrain the *same* mass point, we need to employ constraint satisfaction algorithms

# Time-Corrected Verlet Integration

- Big assumption in basic Verlet: time-delta's are **constant!**
- Solution for non-constant  $\Delta t$ 's:
  - Time steps are:  $t_i = t_{i-1} + \Delta t_{i-1}$  and  $t_{i+1} = t_i + \Delta t_i$

- Expand Taylor series in both directions:

$$\mathbf{x}(t_i + \Delta t_i) \quad \text{and} \quad \mathbf{x}(t_i - \Delta t_{i-1})$$

- Divide the expansions by  $\Delta t_i$  and  $\Delta t_{i-1}$ , respectively, then add both, like in the derivation of the basic Verlet
- Rearranging and omitting higher-order terms yields:

$$\mathbf{x}(t_i + \Delta t_i) = \mathbf{x}(t_i) + \frac{\Delta t_i}{\Delta t_{i-1}} (\mathbf{x}(t_i) - \mathbf{x}(t_i - \Delta t_{i-1})) + \ddot{\mathbf{x}}(t_i) \frac{\Delta t_i + \Delta t_{i-1}}{2} \cdot \Delta t_i$$

- Note: basic Verlet is a special case of time-corrected Verlet



# Implicit Integration (a.k.a. Backwards Euler)

- All explicit integration schemes are only *conditionally stable*
  - I.e.: they are only stable for a specific range for  $\Delta t$
  - This range depends on the stiffness of the springs
- Goal: *unconditionally stability*
- One option: **implicit Euler integration**

explicit

$$\mathbf{x}_i^{t+1} = \mathbf{x}_i^t + \Delta t \mathbf{v}_i^t$$
$$\mathbf{v}_i^{t+1} = \mathbf{v}_i^t + \Delta t \frac{1}{m_i} \mathbf{f}(\mathbf{x}^t)$$

implicit

$$\mathbf{x}_i^{t+1} = \mathbf{x}_i^t + \Delta t \mathbf{v}_i^{t+1}$$
$$\mathbf{v}_i^{t+1} = \mathbf{v}_i^t + \Delta t \frac{1}{m_i} \mathbf{f}(\mathbf{x}^{t+1})$$

- Now we've got a system of non-linear, algebraic equations, with  $\mathbf{x}^{t+1}$  and  $\mathbf{v}^{t+1}$  as unknowns on **both** sides → **implicit integration**



- Write all the implicit equations as **one big** system of equations :

$$M\mathbf{v}^{t+1} = M\mathbf{v}^t + \Delta t \mathbf{f}(\mathbf{x}^{t+1}) \quad (1)$$

$$\mathbf{x}^{t+1} = \mathbf{x}^t + \Delta t \mathbf{v}^{t+1} \quad (2)$$

- Plug (2) into (1) :

$$M\mathbf{v}^{t+1} = M\mathbf{v}^t + \Delta t \mathbf{f}(\mathbf{x}^t + \Delta t \mathbf{v}^{t+1}) \quad (3)$$

- Expand  $\mathbf{f}$  as Taylor series:

$$\begin{aligned} \mathbf{f}(\mathbf{x}^t + \Delta t \mathbf{v}^{t+1}) &= \mathbf{f}(\mathbf{x}^t) + \frac{\partial}{\partial \mathbf{x}} \mathbf{f}(\mathbf{x}^t) \cdot (\Delta t \mathbf{v}^{t+1}) \\ &\quad + O((\Delta t \mathbf{v}^{t+1})^2) \end{aligned} \quad (4)$$

- Plug (4) into (3): 
$$M\mathbf{v}^{t+1} = M\mathbf{v}^t + \Delta t \left( \mathbf{f}(\mathbf{x}^t) + \underbrace{\frac{\partial}{\partial \mathbf{x}} \mathbf{f}(\mathbf{x}^t)}_K \cdot (\Delta t \mathbf{v}^{t+1}) \right)$$
  

$$= M\mathbf{v}^t + \Delta t \mathbf{f}(\mathbf{x}^t) + \Delta t^2 K \mathbf{v}^{t+1}$$

- $K$  is the Jacobi-Matrix, i.e., the derivative of  $\mathbf{f}$  wrt.  $\mathbf{x}$ :

$$K = \begin{pmatrix} \frac{\partial}{\partial x_0} f_0 & \frac{\partial}{\partial x_1} f_0 & \dots & \frac{\partial}{\partial x_{3n-1}} f_0 \\ \vdots & \vdots & \ddots & \vdots \\ \frac{\partial}{\partial x_0} f_{3n-1} & \dots & \dots & \frac{\partial}{\partial x_{3n-1}} f_{3n-1} \end{pmatrix}$$

- $K$  is called the **tangent stiffness matrix**
- (The normal stiffness matrix is evaluated at the equilibrium of the system; here, the matrix is evaluated at an arbitrary "position" of the system in phase space, hence the name)

- Now reorder terms :

$$(M - \Delta t^2 K) \mathbf{v}^{t+1} = M \mathbf{v}^t + \Delta t \mathbf{f}(\mathbf{x}^t)$$

- Now, this has the form:

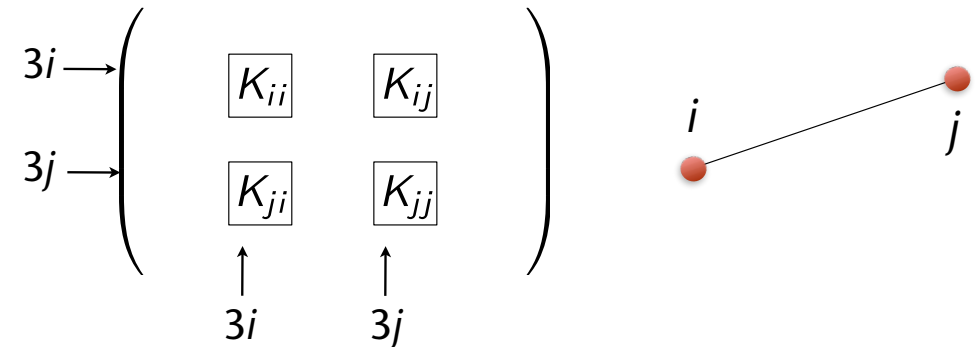
$$A \mathbf{v}^{t+1} = \mathbf{b}$$

$$\text{mit } A \in \mathbb{R}^{3n \times 3n}, \quad b \in \mathbb{R}^{3n}$$

- Solve this system of linear equations with any of the standard iterative solvers
- Don't use a non-iterative solver, because
  - A changes with every simulation step
  - We can "warm start" the iterative solver with the solution as of last frame
    - Incremental computation

# Computation of the Stiffness Matrix

- First of all, understand the anatomy of matrix  $K$  :
  - A spring  $(i, j)$  adds the following four  $3 \times 3$  block matrices to  $K$  :



- Block matrix  $K_{ij}$  arises from the derivation of  $\mathbf{f}_i = (f_{3i}, f_{3i+1}, f_{3i+2})$  wrt.  $\mathbf{x}_j = (x_{3j}, x_{3j+1}, x_{3j+2})$ :

$$K_{ij} = \begin{pmatrix} \frac{\partial}{\partial x_{3j}} f_{3i} & \frac{\partial}{\partial x_{3j+1}} f_{3i} & \frac{\partial}{\partial x_{3j+2}} f_{3i} \\ \vdots & \vdots & \vdots \\ \frac{\partial}{\partial x_{3j}} f_{3i+2} & \dots & \frac{\partial}{\partial x_{3j+2}} f_{3i+2} \end{pmatrix}$$

- In the following, consider only  $f^s$  (spring force)

- First of all, compute  $K_{ii}$ :

$$\begin{aligned} K_{ii} &= \frac{\partial}{\partial \mathbf{x}_i} f_i(\mathbf{x}_i, \mathbf{x}_j) \\ &= k_s \frac{\partial}{\partial \mathbf{x}_i} \left( (\mathbf{x}_j - \mathbf{x}_i) - l_0 \frac{\mathbf{x}_j - \mathbf{x}_i}{\|\mathbf{x}_j - \mathbf{x}_i\|} \right) \\ &= k_s \left( -I - l_0 \frac{-I \cdot \|\mathbf{x}_j - \mathbf{x}_i\| - (\mathbf{x}_j - \mathbf{x}_i) \cdot \frac{(\mathbf{x}_j - \mathbf{x}_i)^\top}{\|\mathbf{x}_j - \mathbf{x}_i\|}}{\|\mathbf{x}_j - \mathbf{x}_i\|^2} \right) \\ &= k_s \left( -I + l_0 \frac{1}{\|\mathbf{x}_j - \mathbf{x}_i\|} I + \frac{l_0}{\|\mathbf{x}_j - \mathbf{x}_i\|^3} (\mathbf{x}_j - \mathbf{x}_i)(\mathbf{x}_j - \mathbf{x}_i)^\top \right) \end{aligned}$$

- Reminder:

- $$\left(\frac{f}{g}\right)' = \frac{f'g - fg'}{g^2}$$

- $$\frac{\partial}{\partial \mathbf{x}} \|\mathbf{x}\| = \frac{\partial}{\partial \mathbf{x}} \left( \sqrt{x_1^2 + x_2^2 + x_3^2} \right) = \frac{\mathbf{x}^T}{\|\mathbf{x}\|}$$



- Using some symmetries, we can analogously derive:

- $K_{ij} = \frac{\partial}{\partial \mathbf{x}_j} f_i(\mathbf{x}_i, \mathbf{x}_j) = -K_{ii}$

- $K_{jj} = \frac{\partial}{\partial \mathbf{x}_j} f_j(\mathbf{x}_i, \mathbf{x}_j) = \frac{\partial}{\partial \mathbf{x}_j} (-\mathbf{f}_i(\mathbf{x}_i, \mathbf{x}_j)) = K_{ii}$

- $K_{ji} = K_{ij}$

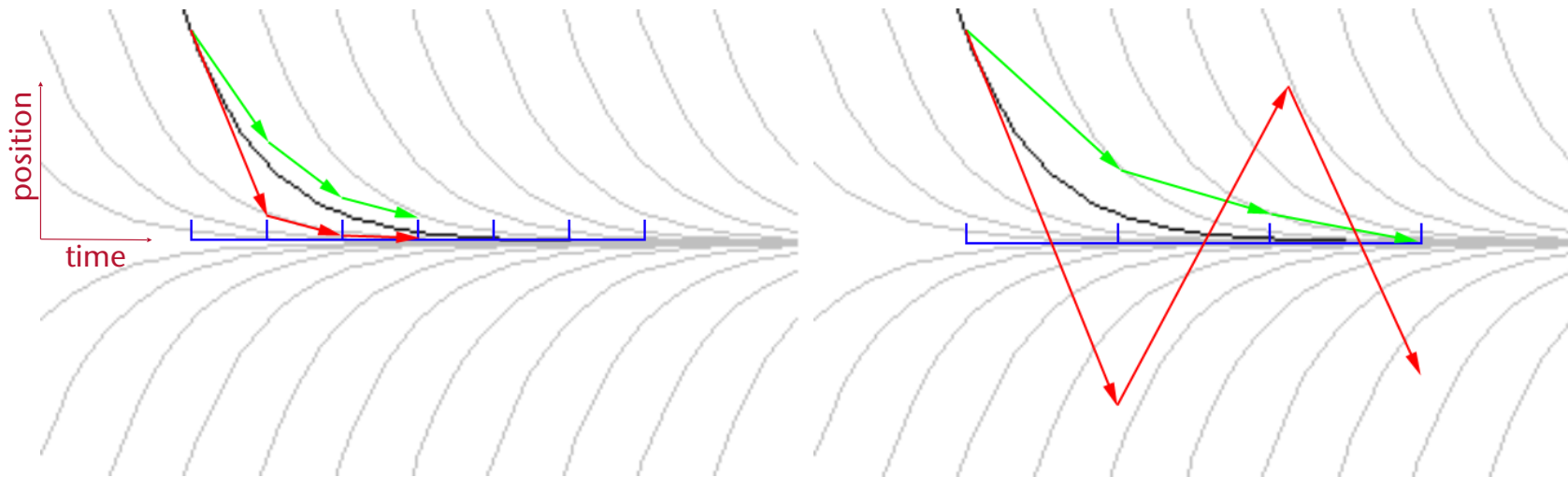
# Overall Algorithm for Solving Implicit Euler Integration

- Initialize  $K = 0$
- For each spring  $(i, j)$  compute  $K_{ii}, K_{ij}, K_{ji}, K_{jj}$  and accumulate it into  $K$  at the right places
 
$$\begin{pmatrix} \boxed{K_{ii}} & \boxed{K_{ij}} \\ \boxed{K_{ji}} & \boxed{K_{jj}} \end{pmatrix}$$
- Compute  $\mathbf{b} = M\mathbf{v}^t + \Delta t\mathbf{f}(\mathbf{x}^t)$
- Solve the linear equation system  $A\mathbf{v}^{t+1} = \mathbf{b} \rightarrow \mathbf{v}^{t+1}$
- Compute  $\mathbf{x}^{t+1} = \mathbf{x}^t + \Delta t\mathbf{v}^{t+1}$

# Advantages and Disadvantages

- Explicit integration:
  - ✓ Very easy to implement
  - Small step sizes needed
  - Stiff springs don't work very well
  - Forces are propagated only by one spring per time step
- Implicit Integration:
  - ✓ Unconditionally stable
  - ✓ Stiff springs work better
  - ✓ Global solver → forces are being propagated throughout the whole spring-mass system within one time step
  - Large time steps needed, b/c one step is much more expensive (if real-time is needed)
  - The integration scheme introduces damping by itself (might be unwanted)

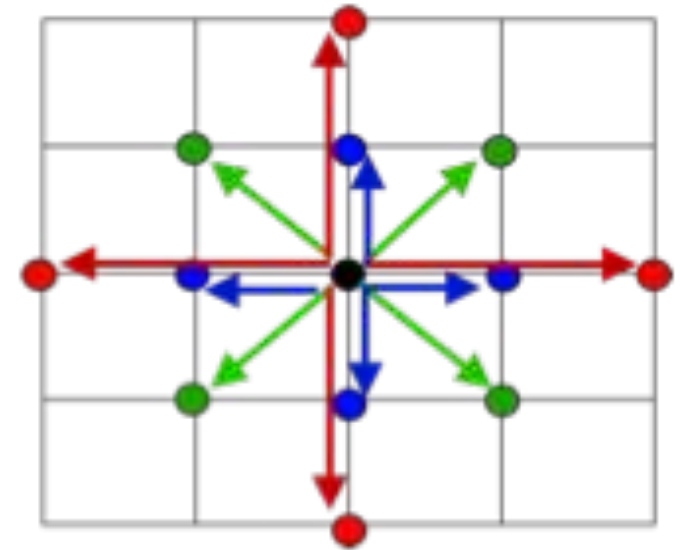
- Visualization of:  $\dot{x}(t) = -x(t)$



- Informal Description:
  - **Explicit** jumps forward blindly, based on current information
  - **Implicit** tries to find a future position and a backwards jump such that the backwards jump arrives exactly at the current point (in phase space)

# Simulating Volumetric Objects

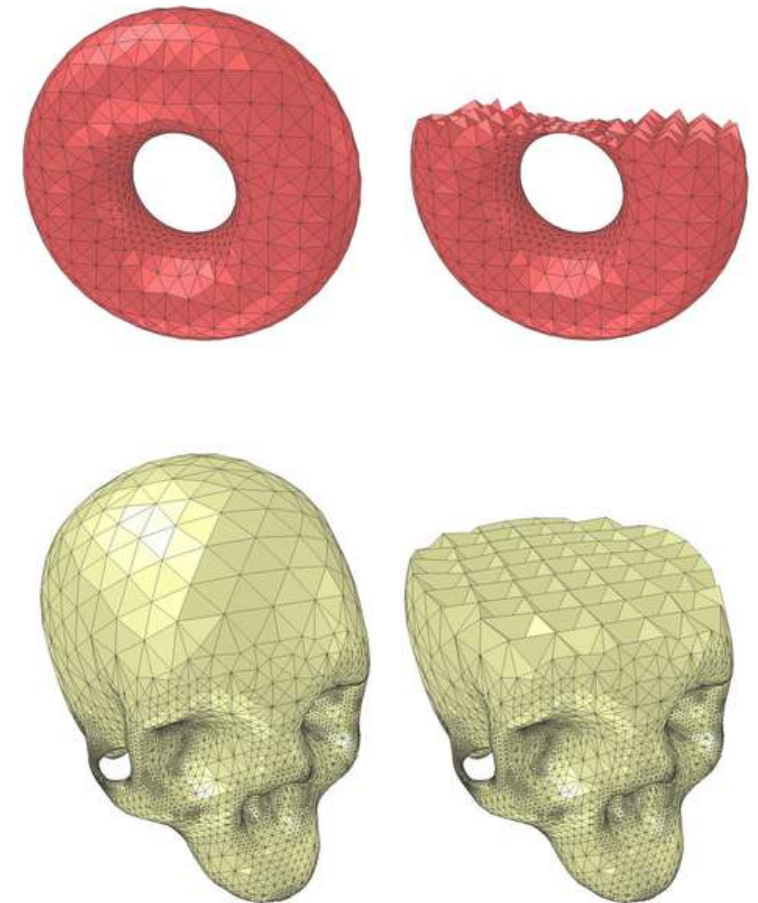
- How to create a mass-spring system for a **volumetric** model?
  - Challenge: volume preservation!
- Approach 1: introduce additional, volume-preserving constraints
  - **Springs** to preserve distances between mass points
  - **Springs** to prevent shearing
  - **Springs** to prevent bending
- No change in model & solver required
- You could also introduce "angle-preserving springs" that exert a **torque** on an edge



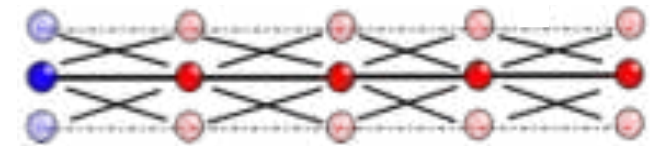
- Approach 2 (and still simple): model the inside volume explicitly
  - Create a tetrahedron mesh out of the geometry
  - Each vertex (node) of the tetrahedron mesh becomes a mass point, each edge a spring
  - Distribute the masses of the tetrahedra ( $= \text{density} \times \text{volume}$ ) equally among the mass points

## Details on Approach 2

- Generation of the tetrahedron mesh (simple method):
  - Distribute a number of points uniformly (perhaps randomly) in the interior of the geometry (so called "**Steiner points**")
  - Dito for a sheet/band outside the surface
  - Connect the points by Delaunay triangulation (see my course "Computational Geometry")
- Variation: create Steiner points outside, too, then anchor the surface mesh within the tetrahedron mesh:
  - Represent each vertex of the surface mesh by the barycentric combination of its surrounding tetrahedron vertices



- Approach 3: kind of an "in-between" between approaches 1 & 2
  - Create a "virtual shell" around the two-manifold (surface) mesh
  - Connect the shell with the "real" mesh by diagonal springs
  
- Video:
  1. no virtual shells,
  2. one virtual shell,
  3. several virtual shells



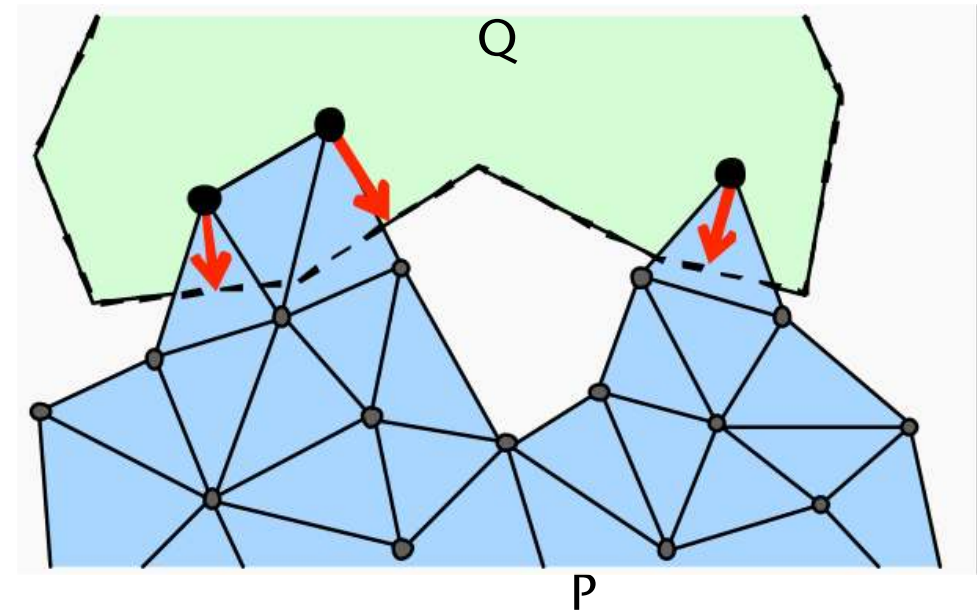


# Collision Detection for Mass-Spring Systems

- Put all tetrahedra in a 3D grid (use a hash table!)
- In case of a collision in the hash table:
  - Compute exact intersection between the 2 involved tetrahedra

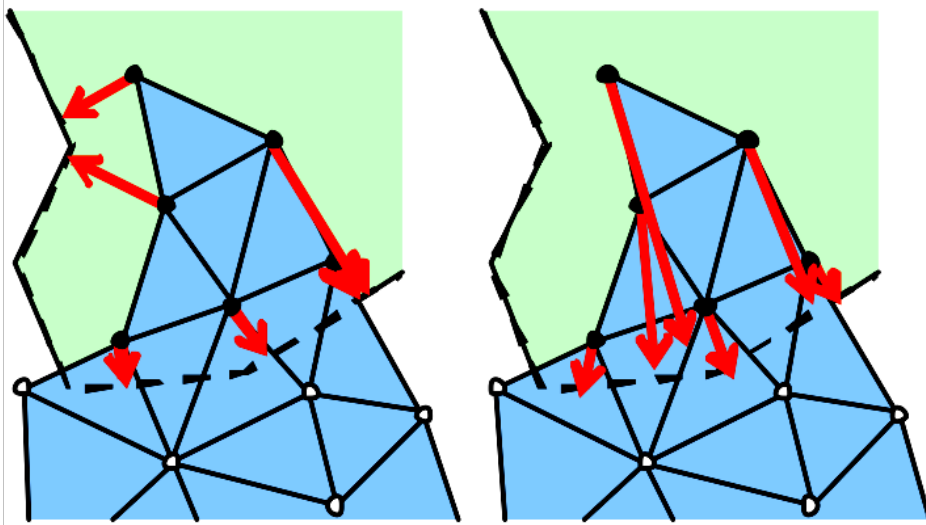
# Collision Response

- Given: objects P and Q (= tetrahedral meshes) that collide
- Task: compute a **penalty force**
- Naïve approach:
  - For each mass point of P that has penetrated, compute its closest distance from the surface of Q → force = amount + direction
- Problem:
  - Implausible forces
  - "**Tunneling**" (s. a. the chapter on force-feedback)



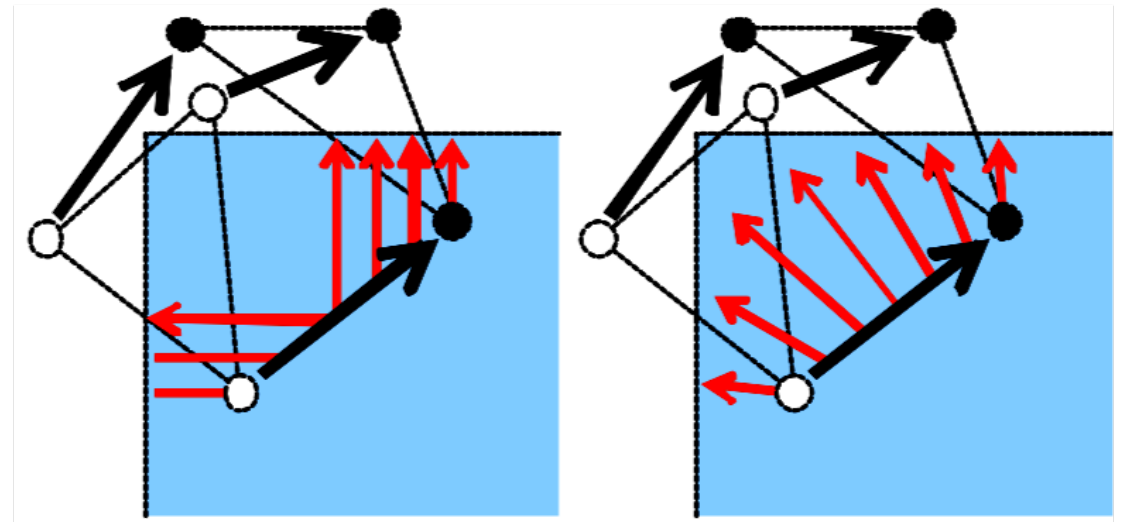
inconsistent

consistent



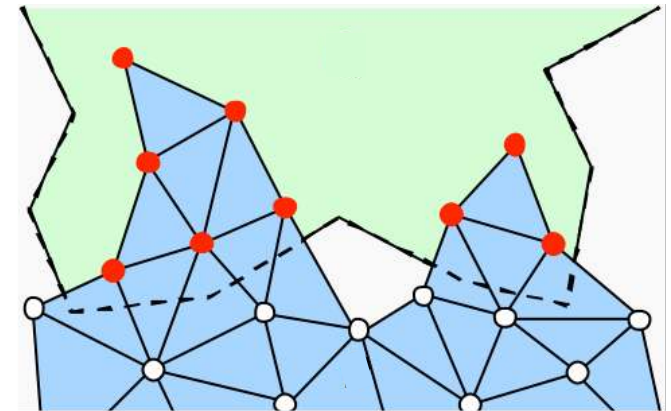
inconsistent

consistent



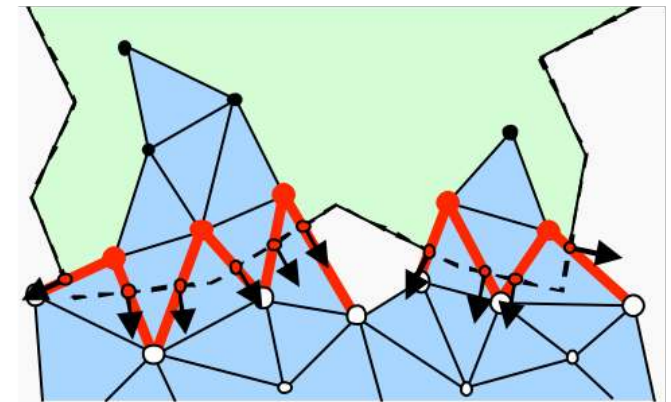
# Consistent Penalty Forces

1. Phase: identify all points of P that penetrate Q



2. Phase: determine all edges of P that intersect the surface of Q

- For each such edge, compute the exact intersection point  $x_i$
- For each intersection point, compute a normal  $\mathbf{n}_i$ 
  - E.g., by barycentric interpolation of the vertex normals of Q

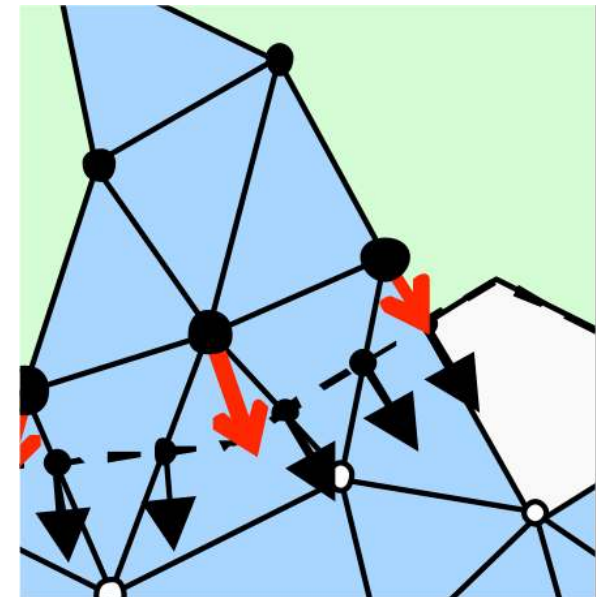


### 3. Phase: compute the approximate force for border points

- Border point = a point  $\mathbf{p}$  that penetrates  $Q$  and is incident to an intersecting edge
- Note: a border point can be incident to several intersecting edges
- Approximate the penetration depth for point  $\mathbf{p}$  by

$$d(\mathbf{p}) = \frac{\sum_{i=1}^k \omega(\mathbf{x}_i, \mathbf{p}) (\mathbf{x}_i - \mathbf{p}) \cdot \mathbf{n}_i}{\sum_{i=1}^k \omega(\mathbf{x}_i, \mathbf{p})}$$

where  $\mathbf{x}_i$  = point of the intersection of an edge incident to  $\mathbf{p}$  with surface  $Q$ ,  
 $\mathbf{n}_i$  = normal to surface of  $Q$  at point  $\mathbf{x}_i$ ,  
 and  $\omega(\mathbf{x}_i, \mathbf{p}) = \frac{1}{\|\mathbf{x}_i - \mathbf{p}\|}$



- Set the direction of the penalty force on border points:

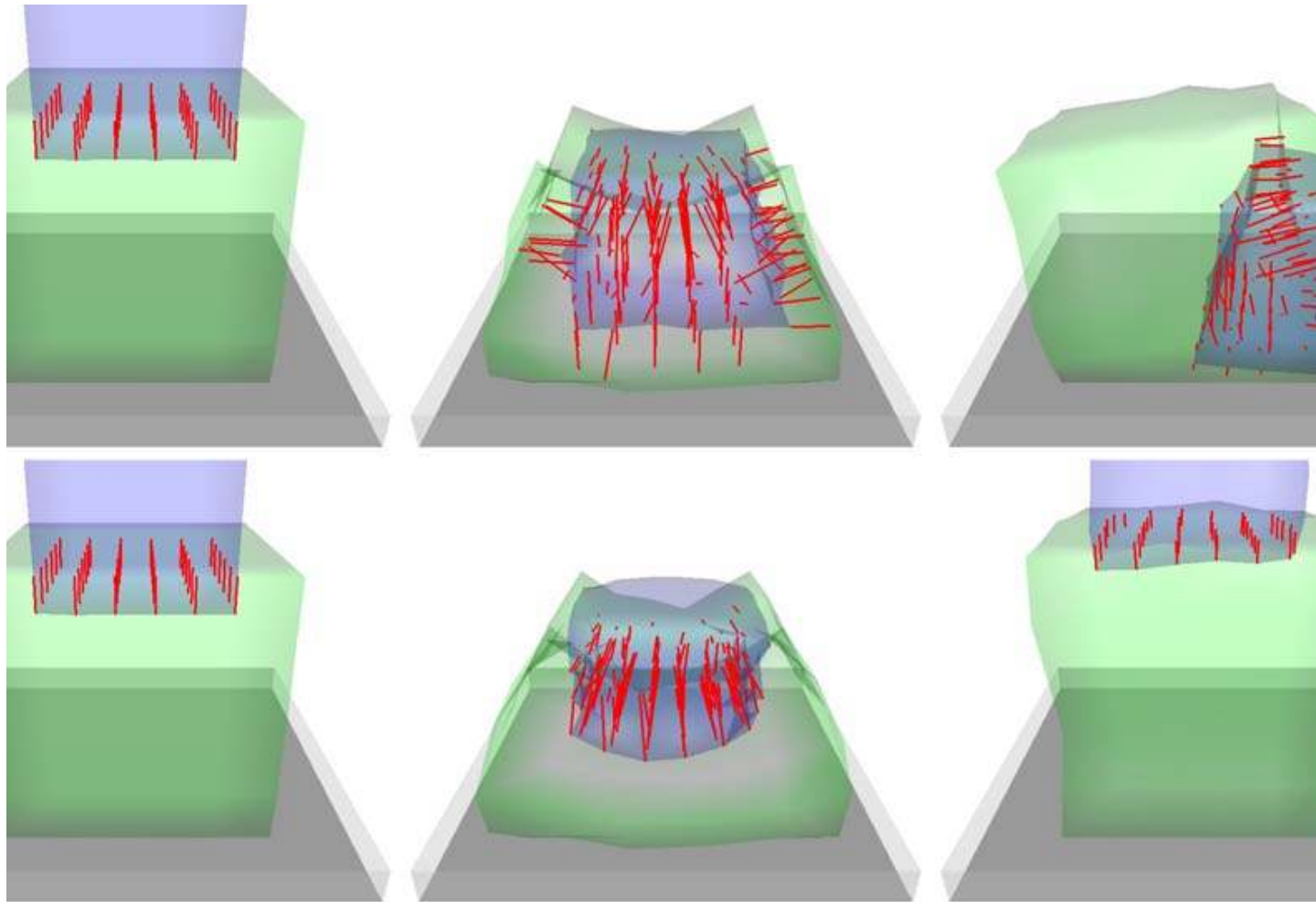
$$\mathbf{r}(\mathbf{p}) = \frac{\sum_{i=1}^k \omega(\mathbf{x}_i, \mathbf{p}) \mathbf{n}_i}{\sum_{i=1}^k \omega(\mathbf{x}_i, \mathbf{p})}$$

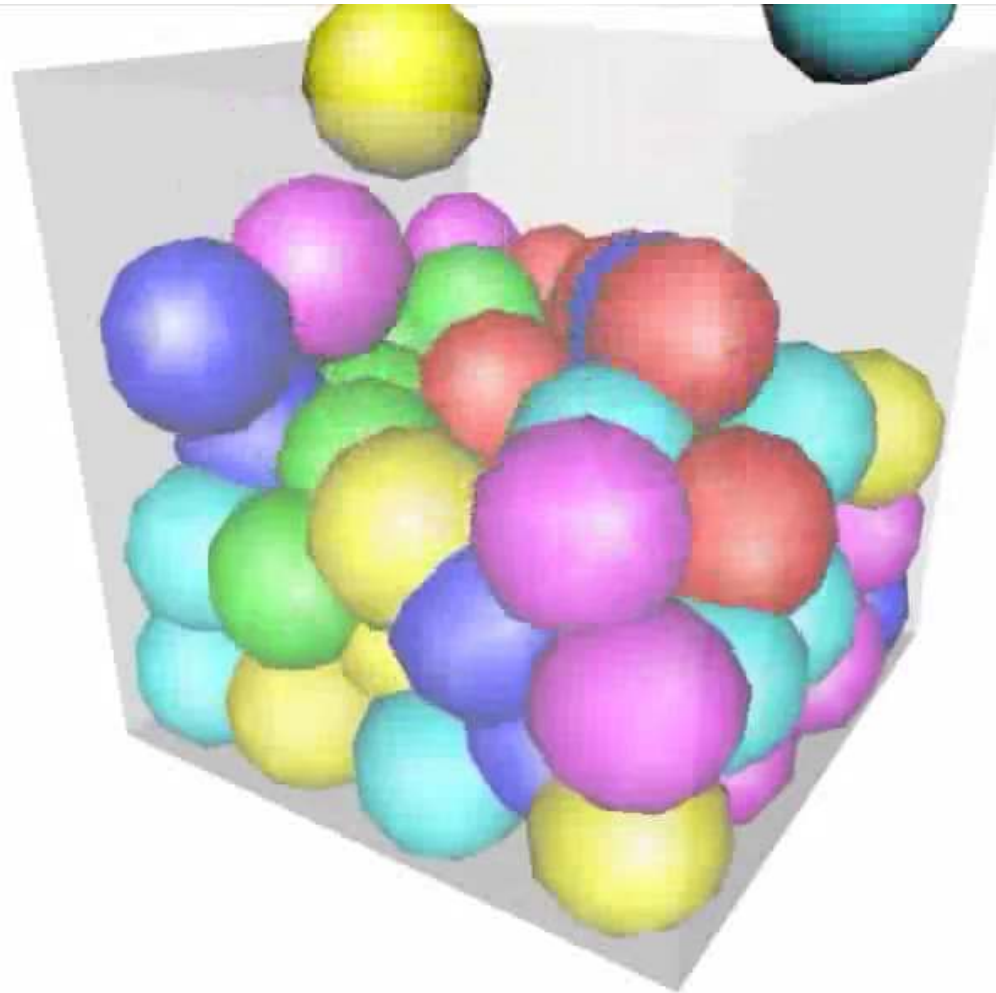
4. Phase: propagate forces by way of breadth-first traversal through the tetrahedron mesh

$$d(\mathbf{p}) = \frac{\sum_{i=1}^k \omega(\mathbf{p}_i, \mathbf{p}) ((\mathbf{p}_i - \mathbf{p}) \cdot \mathbf{r}_i + d(\mathbf{p}_i))}{\sum_{i=1}^k \omega(\mathbf{x}_i, \mathbf{p})}$$

where  $\mathbf{p}_i$  = points of  $P$  that have been visited already,  $\mathbf{p}$  = point not yet visited,  $\mathbf{r}_i$  = direction of the estimated penalty force in point  $\mathbf{p}_i$  .

# Visualization





<http://cg.informatik.uni-freiburg.de>



# Art(?) with Mass-Spring Systems



